

1 JACK

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3 INCORPORATION BY REFERENCE

4
5 The inventors incorporate herein by reference any and all U. S.
6 patents, U. S. patent applications, and other documents cited or
7 referred to in this application or cited or referred to in the U. S. patents
8 and U. S. patent applications incorporated herein by reference.
9

10 DEFINITIONS

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12 The words “comprising,” “having,” and “including,” and other
13 forms thereof, are intended to be equivalent in meaning and be open
14 ended in that an item or items following any one of these words is not
15 meant to be an exhaustive listing of such item or items, or meant to be
16 limited to only the listed item or items.

17 "Rectangular-shape" includes square-shape.
18

19 BACKGROUND OF INVENTION

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21 United States Patent 6,561,487 discloses a personal vehicle jack
22 having a platform for lifting a personal vehicle such as a motorcycle,
23 all terrain vehicle (ATV), or personal watercraft. The jack is designed to
24 lift the entire vehicle off the floor or ground, with the vehicle balanced
25 on a platform. This jack has stabilizing arms connected to a base to
26 provide side-to-side stability, i.e. to prevent tipping over sideways, and
27 lifting arms for elevating the platform in response to manual actuation
28 of a hydraulic cylinder that operates a substantially vertically
29 orientated ram. A user actuates the jack by stepping on a foot pedal.

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Without limiting the scope of this invention as expressed by the claims that follow, some, but not necessarily all, of its features are:

Two, the jack includes a base that may be cast of metal, for example, aluminum. The base may have a pair of spaced apart, unitary, rigid, substantially planar side members each with a lower edge, at least a portion of each lower edge being adapted to rest on ground during use of the jack.

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may be in substantial alignment and the rear ends may be in substantial alignment. The rear segments may be substantially parallel, separated by a second predetermined distance that is less than the first predetermined distance. The rear segments may be in substantial registration. A stiffening element, for example, an axle may extend between the forward ends of the forward segments, and the axle may carry a pair of wheels that lie outboard of the side members. The rear segments may also include one or more wheels.

Four, each side member may include an intermediate segment between its forward and rear segments. The intermediate segments may slant inward towards each other to connect the forward segment and rear segment of each side member. The side members may be mirror images of each other.

Five, the base may have a length of from about 30 to about 40 inches and the forward segments may comprise at least about 50 percent of the length of the base and the rear segments comprise no more than about 50 percent of the length of the base. The intermediate segments may comprise no more than about 25 percent of the length of the base.

Six, the jack includes a substantially horizontally oriented platform that may be cast metal, for example, aluminum. The platform may have a forward end, a rear end, opposed sides, and an upper surface adapted to support a load in an elevated position with the entire load above ground level. The platform may have a width that is substantially equal to the predetermined distance between the forward segments and a length that is substantially equal to the predetermined length of the forward segments. The platform may include a marginal frame with a hollow interior. This platform may have a substantially rectangular-shaped configuration with dimensions that are about equal

to or slightly less than the dimensions of a rectangular space defined by the forward segments. For example, this substantially rectangular space situated between the forward segments may have a length from about 10 to about 25 inches and a width from about 10 to about 25 inches.

Seven, a pair of support arms may each be connected between one side member and the platform. Each support arm may have one end pivotably connected to the forward end of the platform and another end pivotably connected to an intermediary portion of a forward segment of the side member to which the support arm is connected.

Eight, a lift arm elevates the platform. The lift arm includes a forward end pivotably connected to the platform. This forward end may be connected to the rear end of the platform at a central portion thereof. The lift arm includes also a rear end pivotably mounted between the rear segments of the side members. The lift arm may be positioned lengthwise along a longitudinal axis of the jack.

Nine, a driver assembly actuates the lift arm. This driver assembly may be mounted to the base between the rear segments of the side members. The driver assembly may include a hydraulic cylinder having ram element coupled to the lift arm. The ram element in response to manual actuation moves substantially horizontal, causing the platform to move between a lowered position and a plurality of different elevated positions.

Ten, the support arms and lift arm move in unison and substantially parallel to each other so said platform maintains a substantially horizontal orientation as it moves between lowered and elevated positions.

Eleven, the jack may include a detachable, elongated safety stop

member that is manually detached and, when in an elevated position, is located so that at least a portion thereof engages a top edge of the base if the platform abruptly returns to the lowered position. In other words, the drive assembly fails, and the platform rapidly falls towards the ground, the safety stop member breaks this fall.

These features are not listed in any rank order nor is this list intended to be exhaustive.

DESCRIPTION OF DRAWING

Some embodiments of this invention, illustrating all its features, will now be discussed in detail. These embodiments depict the novel and non-obvious jack of this invention as shown in the accompanying drawing, which is for illustrative purposes only. This drawing includes the following figures (Figs.), with like numerals indicating like parts:

Fig. 1 is a left hand perspective view of a jack according to one embodiment of this invention.

Fig. 2 is a right hand perspective view of the jack shown in Fig. 1.

Fig. 3 is a top plan view of the jack in Fig. 1.

Fig. 4 is a side view of the jack in Fig. 1.

Fig. 5 is a cross-sectional view taken along line 5-5 in Fig. 3 illustrating movement of the support arms, platform, lift arm, and handle.

Fig. 5A is an enlarged, fragmentary view taken along line 5A in Fig. 5.

Fig. 6 is an exploded, perspective view of the jack shown in Fig. 2.

Fig. 7 is a perspective view of a drive assembly according to one embodiment of this invention.

1 Fig. 8 is an exploded, perspective view of the drive assembly
2 shown in Fig. 7.

3 Fig. 9 is a perspective view of a grip pad according to an
4 embodiment of this invention showing the underside of the grip pad.

5 Fig. 10 is a perspective view of a support arm according to an
6 embodiment of this invention showing the underside of the support
7 arm.

8 Fig. 11 is a perspective view showing the underside of the jack
9 depicted in Fig. 1, with one side of the base removed.

10 Fig. 12 is a perspective view of the lift arm according to an
11 embodiment of this invention.

12 13 DETAILED DESCRIPTION OF SOME EMBODIMENTS OF THIS INVENTION 14

15 As shown in Figs. 1-4 and 6, one embodiment of this invention, a
16 jack 10, includes a base 12, a horizontally oriented platform 18 when
17 the jack is resting on ground that overlies a front portion of the base, a
18 pair of support arms 114, 116 connecting to the platform and base,
19 and a lift arm 14 connected between the platform and a drive assembly
20 16 mounted at or near a rear portion of the base. In accordance with
21 one feature of this invention, the jack is light-weight, weighing less
22 than about 85 pounds, and is designed to lift a light weight load such
23 as a personal vehicle completely off the ground. Typically, the load
24 does not exceed about 2500 pounds. Moreover, to reduce costs,
25 standard components are used such as the drive assembly 16,
26 commonly used in floor jacks that lift, for example, one end of an
27 automobile but are not suited to lift a personal vehicle completely off
28 the ground. To reduce weight, the base 12, platform 18, and lift arm
29 14 are cast from aluminum. Using such cast metal components not only

1 reduces weight, but also eliminates many parts commonly found in
2 conventional personal vehicle jacks.

3 As illustrated best in Fig. 6, the base 12 includes two separate
4 components, a left side 80 and a right side 90, that are substantially
5 mirror images of each other. Each side 80, 90 has a forward
6 substantially planar segment 82, 92, a substantially planar rear
7 segment 84, 94 that lies inward of the forward segment, and a
8 substantially planar intermediate segment 86, 96 that connects the
9 forward and rear segments. There are substantially triangular shaped
10 outer braces 88, 98 (Fig. 3) integral with exteriors of the sides 80, 90,
11 respectively, the brace 88 extending along the exterior of the
12 intermediate segment 86 and its adjacent rear segment 84 and the
13 brace 98 extending along the exteriors of the intermediate segment 96
14 and its adjacent rear segment 94. As best illustrated in Fig. 3 and 11,
15 there is a substantially wedge shaped inner brace 87, 97 integral with
16 the interiors of each side 80, 90, respectively. The inner braces 87, 97
17 each comprise a block having a triangular portion 87a, 97a, a
18 rectangular portion 87b, 97b integral with triangular portion, and a
19 flange 87c, 97c. As best depicted in Fig. 11, each flange 87c, 97c along
20 with an adjacent portion of a side 80, 90, as the case may be, form a
21 yoke Y1. There are holes 182a in each of these flanges 87c, 97c that
22 are aligned with each other and with adjacent holes 182 in the sides
23 80, 90. The rectangular portions 87b, 97b are integral with the
24 forward segments 82, 92 (Fig. 3) and the triangular portions 87a, 97a
25 (Fig. 3) are integral with the intermediate segments 86, 96,
26 respectively. The flanges 87c, 97c may be located at about the
27 midpoint of the left 80 and right 90 sides, respectively.

28 As illustrated best in Figs. 3 and 6, the forward segments 82, 92
29 are parallel, of equal lengths, in registration, and equidistance from the

1 longitudinal axis X (Fig. 3) of the jack 10. Each forward segment 82, 92
2 forms a substantially vertical wall when the jack 10 is resting on
3 ground, with a hole 82a, 92a (Fig. 6) nearby the fronts 82b, 92b (Fig.
4 6), respectively, a hole 180 (only one shown) nearby the intermediate
5 segment 86, 96, respectively. As shown in Fig. 3, the forward segments
6 82 and 92 lie outward O of the platform 18 where the distance between
7 the forward segments is slightly greater than the width w_1 (Fig. 3) of
8 the platform. The distance d_1 between the forward segments 82, 92 is
9 from about 10 to about 25 inches and the length l_1 of each forward
10 segments 82, 92 is from about 10 to about 25 inches. These
11 dimensions define a rectangular area over which the platform 18 lies
12 and the platform may be substantially rectangular and have
13 dimensions about equal to or slightly less (no more than about 5
14 percent) than this area.

15 As illustrated best in Figs. 3 and 6, the rear segments 84 and 94,
16 which are parallel and of equal lengths and in registration. Each form
17 a substantially vertical wall when the jack 10 is resting on ground, with
18 a hole 84a, 94a nearby the fronts 84b, 94b (Fig. 6) and tops 84c, 94c
19 (Fig. 6) of each segment and another hole 84d, 94d nearby the middle
20 bottom 84e, 94e of each segment. The rear segments 84 and 94 each lie
21 laterally between the forward segments 82, 92 and straddle the
22 longitudinal axis X of the jack 10. Each rear segment may be
23 equidistance from this axis, typically from about 5 to about 10 inches
24 from the longitudinal axis X. The intermediate segments 86 and 96 may
25 slant inward towards each other to connect the forward segments 82,
26 92 and rear segments 84, 94, respectively. These intermediate
27 segments 86 and 96 form substantially vertical walls and they have
28 equal lengths from about 8 to about 12 inches. The forward segments
29 82 and 92, intermediate segments 86 and 96, and rear segments 84

1 and 94 may slope upward from the forward to rear segments to
2 increase gradually in height. The height of these segments typically
3 ranges from about 3 to about 7 inches.

4 As shown best in Figs. 1 and 6, the platform 18, which may be
5 cast from aluminum, comprises (a) a substantially rectangular,
6 horizontally oriented, rectangular frame 48 having a pair of yokes 50,
7 52 each near a front corner of the platform and extending from an
8 underside 48e (Fig. 11) of the forward end 48a of the platform, (b) a
9 central, rectangular shaped opening 54, (c) pair of opposed sides 48c,
10 48d, and (d) a yoke Y2 (Fig. 11) including a pair of opposed, parallel
11 walls 60, 62 extending along the underside 48e of the platform inward
12 from the rear end 48b of the platform. An open end 50a, 52a (Fig. 1)
13 of each yoke 50, 52 faces downward, and a pair of arms 50b, 50c and
14 52b, 52c (Fig. 6) of each yoke has a hole 50d, 50e and 52d, 52e,
15 respectively. Each of the sides 48c and 48d has a horizontally
16 orientated hole 56a, 58a near the rear end 48b of the platform 18.
17 Each wall 60 and 62 extends from the rear end 48b of the platform 18
18 to the rectangular opening 54 of the platform, and each has a hole 60a,
19 62a that is aligned with the holes 56a, 58a of the outer, opposed sides
20 48c and 48d. These walls 60 and 62 (Figs. 2 and 11) are equidistance
21 from the longitudinal axis X and they are separated by a distance that
22 is substantially equal the width w_2 (Fig. 12) of the forward end of the
23 lift arm 14. This width w_2 ranges from about 3 to about 6 inches. U-
24 shaped tie elements 64 may be attached to the forward end 48a and
25 rear end 48b of the platform 18. Elastic bands (not shown) are
26 wrapped or tie to these tie elements 64 (Fig. 1) and the vehicle being
27 balanced on the platform 18 to hold the vehicle securely to the
28 platform.

29 A pair of laterally adjustable grips pads 64 (Figs. 1 and 6) may be

1 connected to the top side 18a of the platform 18. As illustrated in Fig.
2 9, each grip pad 64 comprises a metal plate 66 with a coating 68
3 preferably made from a non-slippery substance such as rubber applied
4 to the top side 66a of the metal plate, and a pair of spaced-apart metal
5 blocks 70, 72 located on the bottom side 66b of each of the metal
6 plates. A threaded cylinder 70a, 72a extends outward from each of the
7 metal blocks 70, 72, respectively. The grip pads 64 may be coupled to
8 the platform 18 by inserting the threaded cylinders 70a, 72a through
9 slots 74a, 74b, 74c, 74d of the platform, respectively, and attaching a
10 nut (not shown) to each of the threaded cylinders. The location of each
11 of the grip pads 64 on the platform 18 may be varied by sliding the
12 threaded cylinders 70a, 72a along the slots 74a, 74b, 74c, 74d until a
13 desired position is achieved. This provides more or less exposure of
14 the rectangular opening 54 as may be need to accommodate the
15 undercarriage of a vehicle being supported by the platform 18 or to
16 better balance the vehicle on the platform.

17 Wheels 100, 102, 104 and 106 may be attached to the base 12. A
18 stiffening rod 108, also functioning as an axle, may be attached to the
19 left side 80 and right side 90 of the base 12 by passing a left end 108a
20 and right end 108b of the rod through holes 82a, 92a, respectively. A
21 secondary stiffening rod 107 may also extended between the left side
22 80 and right side 90 nearby the junctions between the forward
23 segments 82 and 92 and the intermediate segments 86 and 96 of these
24 sides. The front wheels 100, 102 may be attached to the rod 108
25 outboard of the left side 80 and right side 90. Referring to Fig. 3, the
26 front wheels 100, 102 also each lie outward of the platform 18. The
27 rear wheels 104, 106 are caster type wheels and may be detachably
28 connected to the rear segments 84, 94 of the base 12 by wheel mounts
29 110, 112 (Fig. 6). These wheel mounts 110, 112 are screwed or

1 otherwise attached to the outer sides 80a, 90a of the rear segments 84,
2 94 of the base, respectively. In another embodiment, the wheel mounts
3 may be integral (not detachable) with the rear segments 84, 94 of the
4 base 12. Referring to Fig. 3, each of the wheel mounts 110, 112 lie
5 inside of the forward segments 82, 92 of the base 12 but are outboard
6 of the rear segments 84, 94.

7 As shown best in Figs. 2, 6 and 10, a pair of support arms 114
8 and 116 each have opposed ends pivotably connected to the base 12
9 and platform 18. The support arms 114, and 116 each comprise an
10 elongated bar having horizontally, orientated holes 114a, 114b (Fig.
11 10) and 116a, 116b at opposed ends 114c, 114d and 116c, 116d,
12 respectively. A cylindrical stop member 118, 120 may be located in a
13 channel 114e, 116e of each support arm 114, 116, nearby ends 114d
14 and 116d, respectively. Pivot pins P1 (Fig. 5) extend through holes
15 114a, 116a in the support arms 114, 116 and the holes 50d, 50e and
16 52d, 52e in the yokes 50, 52 along the forward end 48b of the platform
17 18. In a similar manner, pivot pins P2 (Fig. 5) extend through holes
18 114b and 116b and the holes 180 in the sides 80,90 and the holes
19 182a in the flanges 87c, 97c. Each support arm 114, 116 is thus
20 pivotably connected at opposed ends to the platform 18 and
21 intermediary portions of the forward segments 82,92 of the base 12.

22 Referring to Figs. 6, 11 and 12, the lift arm 14 is a rigid, unitary
23 member that may be cast from aluminum. It is connected to pivot at
24 its opposed forward end 14a and rear end 14b respectively to the
25 platform 18 and the drive assembly 16. The lift arm 14 includes a left
26 triangular wall 124 and a right triangular wall 122 that are
27 substantially parallel. It also includes a front connector 126 at the
28 forward end 14a, a middle connector 128, and rear connector 130 at
29 the rear end 14b; all extending between the walls 122 and 124

1 substantially at a right angle. These triangular walls 122, 124 each
2 have a horizontally orientated hole 125, 127 near the front ends 122a,
3 124a aligned with each other, a horizontally orientated hole 146, 148,
4 near the rear of these walls aligned with each other, and a horizontally
5 orientated hole 200, 202 between the middle connector 128 and the
6 rear end 14b of the lift arm.

7 The rear connector 130 provides a housing for the drive
8 assembly 16. As best shown in Figs. 11 and 12, the rear connector 130
9 includes a top plate 132 (Figs. 1 and 2), a rear wall 134 and a parallel
10 front wall 134a, each having concave edges E1 and E2 respectively, and
11 a left sidewall 136 and a right sidewall 138. The top plate 132 is U-
12 shaped and is flush with the top edges of the triangular walls 122 and
13 124. The top plate 132 is open-ended facing forward F. The rear wall
14 134 is U-shaped, having an open end facing towards the bottom sides
15 122d, 124d of the triangular walls 122 and 124. The sidewalls 136 and
16 138 are spaced from adjacent portions of the rear segments 84 and 94
17 to provide a space for links 150 and 152 of the drive assembly 16.
18 There is in each sidewall 136,138 a horizontally, orientated hole 140
19 (only one shown in Fig. 12) passing therethrough. The holes 140 in
20 each of these sidewalls 136 and 138 are aligned. There are holes 141
21 (only one shown in Fig. 12) in the triangular walls 122 and 124 that
22 are aligned with the holes 140. A cylindrical boss 144a (only one
23 shown in Fig. 12) projects outward from each of the triangular walls
24 122 and 124 near the rear end 14b and there are holes 144 in each of
25 these bosses that are aligned. As illustrated in Fig. 3, the bosses 144a
26 act as spacers to maintain the rear segments 84,94 and the triangular
27 walls 122, 124 a fixed distance way from each other.

28 As shown in Fig. 3, to connect the forward end 14a of the lift arm
29 14 to a central portion of the rear of the platform 18, the holes 125

1 and 127 at the forward end of the lift arm 14 are aligned with the holes
2 60a, 62a in the walls 60, 62 of the yoke Y2 (Fig. 11) and a pivot pin P3
3 is then inserted into these aligned holes. In an alternate embodiment,
4 the forward end 14a of the lift arm 14 may be pivotably connected to
5 the platform 18 using a rod that passes through holes 125 and 127 of
6 the lift arm, holes 60a and 62a, as well as holes 56a and 58a, of the
7 platform. The rear end 14b of the lift arm 14 is pivotably connected to
8 the base 12 by a dowel 172 that extends through the aligned holes 144
9 in the bosses 144a. The opposed ends 172a and 172b respectively of
10 the dowel 172 are received in the aligned holes 84a and 94a in the rear
11 segments 84 and 94. When the drive assembly 16 actuates the lift arm
12 14, the lift arm pivots about the dowel 172.

13 As depicted in Figs. 5A, 7 and 8, the drive assembly 16 is of a
14 conventional design and includes a ram 19 disposed within a cylinder
15 20, a fluid chamber 22, and a manually operated pump 24. The
16 longitudinal axis of the cylinder 20 is substantially horizontally
17 orientated. The pump 24 is partially disposed within the fluid chamber
18 22, and includes a detachable handle 26, a pump core 28, pump case
19 30, a spring 32, a piston cover 34 and a discharge valve rod 36 for a
20 valve (not shown). The handle 26 is attached to the pump case 30 by a
21 handle base 25. The cylinder 20 is encased in a sleeve 38 and it
22 extends from the front side 22a of the fluid chamber 22. This cylinder
23 20 has, for example, a circular cross-section. The fluid chamber 22 has
24 an internal cavity (not shown) holding hydraulic fluid and a pair of
25 cylindrical caps 40, 42, closing the cavity, each cap having a threaded
26 portion 40a, 42a, respectively, that is used to attach the caps to a main
27 body 44 of the fluid chamber.

28 The main body 44 may be box-like in shape, having a left wall
29 44a and a right wall 44b separated by a distance that is about equal to

1 the distance between the two rear segments 84 and 94 of the base 12.
2 By inserting the caps 40 and 42 into the holes 84d and 94d,
3 respectively, the drive assembly 16 is connected between the rear
4 segments 84 and 94 abutting, respectively, the left wall 44a and right
5 wall 44b (Fig. 8) of the drive assembly 16. A removable fluid plug 46
6 seals an access port 46a that enables fluid to be put into the fluid
7 chamber 22. The ram 19 is mounted to slide forward and rearward
8 within the cylinder 20 and the cross-section of the ram may be
9 identical in shape as the interior I of the cylinder. While one
10 embodiment of a drive assembly 16 is described, other types of drive
11 assemblies may be used such as described in United States Patent Nos.
12 2,629,583, 3,807,694, and 4,018,421. The sleeve 38 abuts the upper
13 edges E1 and E2 of the rear and front walls 134 and 134a, respectively
14 The lift arm 14 is connected to the drive assembly 16 by means
15 of a U-shaped member 157 including a block 154 having a pair of
16 fingers 154a, 154b, each pivotably connected to one of a pair of links
17 150 and 152 that extend towards the main body 44 of the fluid
18 chamber 22. The block 154 is connected to a front end 19a (Fig. 7) of
19 the ram 19. Each link 150, 152 comprises an elongated, rigid bar each
20 having opposed holes 153 and 156, and 158 and 160, respectively. The
21 fingers 154a and 154b, fit into the holes 153 and 158, respectively,
22 with the fingers serving as pivot pins. The other ends of the links 150
23 and 152 are pivotably attached to the rear connector 130. A pivot pin
24 P4 is aligned with the aligned holes 140 and 141 respectively in the left
25 sidewall 136 of the rear connector 130 and right triangular wall 122
26 and these aligned holes are aligned with the hole 156 in the link 150.
27 This pivot pin P4 extends through these aligned holes 140, 141, and
28 156. A pivot pin P5 is aligned with the aligned holes 140 and 141
29 respectively in the right sidewall 138 of the rear connector 130 and left

1 triangular wall 124 and these aligned holes are aligned with the hole
2 160 in the link 152. This pivot pin P5 extends through these aligned
3 holes 140, 141, and 160.

4 Referring to Fig. 5, with the platform 18 in its lowered position
5 shown in dotted lines, the drive assembly 16 is manually actuated to
6 move the platform to one of a plurality of different elevated positions
7 shown in solid lines. The ram 19 is now in a fully retracted condition.
8 To achieve this the user moves the handle 26 first in a downward
9 stroke in a clockwise (CW) direction whereby fluid is moved by the
10 pump 24 from the fluid chamber 22 into the cylinder 20. Moving the
11 handle 26 in an upward stroke in a counter-clockwise (CW) direction
12 does nothing. When fluid enters the cylinder 20, the ram 19 moves
13 outward along the longitudinal axis of the cylinder, pushing the block
14 154 outward towards the main body 44 of the fluid chamber 22,
15 causing the links 150 and 152 to pull on the lift arm 14. This causes the
16 lift arm 14 to pivot about the dowel 172, rotating in a clockwise
17 direction as viewed in Fig. 5 to move the platform 18 from the lowered
18 position to the elevated position. As the lift arm 14 rotates, the support
19 arms 114 and 116 rotate in unison therewith and parallel thereto
20 maintaining the platform 18 substantially horizontal as it is elevated.

21 Repeatedly reciprocating the handle 26 in the clockwise and
22 counter-clockwise direction will continue to elevate the platform 18.
23 Stop members (not shown) are located on the base 12 to limit rotation
24 of the handle 26. To lower the platform 18 to return it to its lowered
25 position shown in dotted lines in Fig. 5, the handle 26 is twisted to
26 actuate the discharge valve rod 36, allowing fluid to move slowly from
27 the cylinder 20 into the fluid chamber 22, with the platform lowering
28 as the fluid returns to the fluid chamber. The handle 26 has a
29 sufficient length to allow a user that is standing upright to actuate the

1 handle without having to significantly adjust his or her posture.

When the platform 18 is elevated, it is desirable to prevent its returning to the lowered position in the event a failure occurs in the drive assembly 16, for example, hydraulic fluid rapidly escaping from the cylinder 20. One way is to provide a safety stop member such as, for example, a detachable, elongated shaft 206 that is mounted to the base 12 for example. With the platform 18 elevated, the shaft 206 is detached and inserted in the aligned holes 200 and 202. If the platform 18 suddenly moves downward because of the failure in the drive assembly 16, the outer ends 206a and 206b of the shaft 206 are located to engage a top edge of the base 12 to prevent the elevated platform from abruptly returning to the lowered position shown in solid lines in Fig. 5.

2 3 SCOPE OF THE INVENTION 4

5 The above presents a description of the best mode contemplated
6 of carrying out the present invention, and of the manner and process
7 of making and using it, in such full, clear, concise, and exact terms as
8 to enable any person skilled in the art to which it pertains to make and
9 use this invention. This invention is, however, susceptible to
10 modifications and alternate constructions from that discussed above
11 which are fully equivalent. Consequently, it is not the intention to
12 limit this invention to the particular embodiments disclosed. On the
13 contrary, the intention is to cover all modifications and alternate
14 constructions coming within the spirit and scope of the invention as
15 generally expressed by the following claims, which particularly point
16 out and distinctly claim the subject matter of the invention: